

# MECH 5810 Module 3: Conservation of Linear Momentum

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MECH 5810 Advanced Fluid Dynamics  
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- 1 Announcements
- 2 Dynamics: Origin of Forces In Fluids
  - Volume Forces
  - Surface Forces
  - Line Forces
- 3 Integral Conservation of Linear Momentum
  - Material volumes
  - General conservation law for material volumes
  - General transport equation for control volumes

# Announcements

- Homework # 1 is posted online.
- Any questions?

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# Fluid Motion 101

- Newton's Laws will govern the motion of a fluid.
- Today's key concept:

$$\sum \vec{F} = \frac{\partial(\text{momentum})}{\partial t} = \frac{\partial(m\vec{u})}{\partial t} = \underbrace{\vec{u} \frac{\partial m}{\partial t}}_{\text{Changing Mass}} + \underbrace{m\vec{a}}_{\text{acceleration}} \quad (1)$$

- First: The forces acting on a control volume.
- Second: The resulting change in momentum of the fluid.

# Formalizing External Forces: Body Forces

- *Body* or *Volume* Forces: result from the fluid being placed in a *force field*:
  - Gravity (Conservative)
  - Electrostatic (Conservative)
  - Magnetic (Conservative)
  - Electromagnetic
- Conservative forces:
  - Work done by the force is independent of the path
  - $\sum(PE + KE) = Const.$
- The body force acts on the volume of fluid (hence does not "touch" the fluid, but acts "on" the fluid)
- eg:

$$\vec{F}(t)_{vol} = \iiint_{MV(t)} \rho \vec{g} dV$$

# Formalizing External Forces: Body Forces

- Representing body forces:

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- **Surface Forces**
- Line Forces

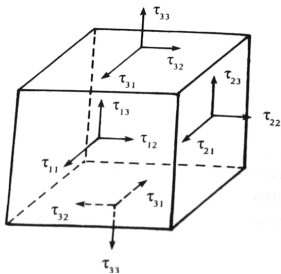
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# Formalizing External Forces: Surface Forces

- Surface forces: Act by direct contact on an area
- Expressed as "per area" quantities
  - Pressure (Normal force per unit area)  $\rightarrow F_N = \iint -p \cdot d\vec{A}$
  - Shear stress (Tangential force per unit area)  $\rightarrow F_t = \iint \tau dA$
- 9 components make up the stress tensor at a given point (more later in the course):



# Formalizing External Forces: Surface Forces

- Representing surface forces due to normal stress/pressure:

# Formalizing External Forces: Surface Forces

- Representing surface forces due to shear stresses:

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# Formalizing External Forces: Line Forces

- Line forces: Act along a contact line.
- Expressed as "per length" quantities
  - Surface tension requires 2 fluids in contact.
  - Surface tension  $\rightarrow F_\sigma = \int \sigma dl$
  - Surface tension acts perpendicular to a cut line.

# Formalizing External Forces: Line Forces

- Representing line forces due to surface tension:

# Formalizing External Forces: Putting it all together

- Determine the NET force on a volume of fluid.
- This is the force that will *produce* a momentum change.

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# Integral Conservation of Linear Momentum: Material Volume

- At **any** material point,  $\frac{\partial(m\vec{u})}{\partial t} = m\vec{a} = \sum \vec{F}$  applies
- For an infinitesimal material volume:

$$m = \rho\delta V = \rho\delta x\delta y\delta z$$

- and:

$$\vec{a} = \frac{\partial\vec{u}}{\partial t}$$

- So, for a single material particle/infinitesimal volume:

$$m\vec{a} = (\rho\delta V) \left( \frac{\partial\vec{u}}{\partial t} \right) = \frac{\partial(\rho\vec{u}\delta V)}{\partial t} = \vec{F}$$

# Integral Conservation of Linear Momentum: Material Volume

- To apply conservation of momentum over the entire *material volume*, we sum (integrate) the infinitesimal contributions of all material particles over the the volume:

$$\frac{d}{dt} \iiint_{MV(t)} \underbrace{\rho \vec{u}}_{\text{momentum per vol}} dV = \sum \vec{F}_{MV}(t)$$

- Recall, that the conservation of mass statement (for MV) is similar:

$$\frac{d}{dt} \iiint_{MV(t)} \rho dV = 0$$

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# Integral Conservation of Linear Momentum: Material Volume

- We can generalize this into a transport law for material volumes, which states:

$$\frac{dB}{dt} = \frac{d}{dt} \iiint_{MV(t)} \rho b dV$$

- Where  $b$  is the *per unit mass* representation of the quantity  $B$ .

Property	$B$	$b$
Mass	$m_s$	1
Linear Momentum	$m_s \vec{u}$	$\vec{u}$
Angular Momentum	$\vec{r} \times m_s \vec{u}$	$\vec{r} \times \vec{u}$
Kinetic Energy	$\frac{1}{2} m_s \ \vec{u}\ ^2$	$\frac{\ \vec{u}\ ^2}{2}$

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# Control Volume Integral Conservation Transport Equation

- Laws of physics are usually/commonly written for a fixed mass of material in a Lagrangian description (or Material Volume)
- We want to be able to calculate:

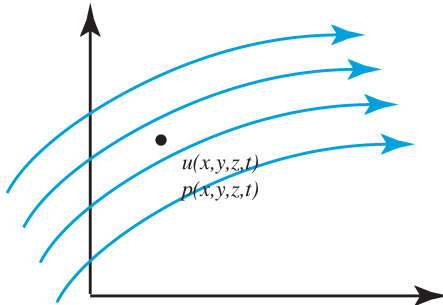
$$\frac{dB}{dt} = \iiint_{MV} \rho \frac{Db}{Dt} dV$$

using an *Eulerian description* of the fluid.

- To do this, we must take a brief detour to understand how to calculate change in a property w.r.t. time.

# ASIDE: Eulerian Description

- Eulerian description: examines fluid properties at an individual stationary point
- Independent variables:  $\vec{r}'$  position in space,  $t'$  time.
  - A fluid property,  $F$  is expressed as  $F(\vec{r}', t')$



## ASIDE: Lagrangian Derivatives in an Eulerian Description(KC section 3.3)

- Lagrangian: time rate of change of a property  $F(\vec{a}, t)$ , is computed as we follow a material point.
- How do we compute this same **important** rate of change in an Eulerian or field description?
  - $\frac{\partial}{\partial t}[F(\vec{r}', t')] \neq \frac{\partial}{\partial t}[F(\vec{a}, t)]$
  - In the Lagrangian we are following the material particles, not staying fixed in space.
- So,

$$\left( \frac{\partial [F(\vec{r}', t')]}{\partial t} \right)_a = ?$$

- I.e. What is the rate of change of a property if we follow a particle, if we were to measure only at a fixed field point?



## **ASIDE: Lagrangian Derivatives in an Eulerian Description(KC section 3.3)**

## ASIDE: Lagrangian Derivatives in an Eulerian Description (KC section 3.3)

- Consider the property  $F$  when  $\vec{r}(\vec{a}, t) = \vec{r}'$  and  $t = t'$ :

$$F(\vec{a}, t) = F[\vec{r}(\vec{a}, t)] = F(\vec{r}', t')$$

- Let's differentiate, careful to use the chain rule:

$$\left[ \frac{\partial F(\vec{a}, t)}{\partial t} \right]_{\vec{a}} = \left( \frac{\partial F}{\partial t'} \right)_{\vec{r}'} \left( \frac{\partial t'}{\partial t} \right) + \left( \frac{\partial F}{\partial \vec{r}'} \right)_{t'} \cdot \left( \frac{\partial \vec{r}'}{\partial \vec{r}} \right) \cdot \left( \frac{\partial \vec{r}}{\partial t} \right)_{\vec{a}}$$

- Review: Chain Rule

$$\frac{\partial z}{\partial t} = \frac{\partial z}{\partial x} \frac{\partial x}{\partial t} + \frac{\partial z}{\partial y} \frac{\partial y}{\partial t} \quad (2)$$

# ASIDE: Lagrangian Derivatives in an Eulerian Description (KC section 3.3)

- Let's look term by term and simplify the equation:

$$\begin{aligned}
 \left[ \frac{\partial F(\vec{a}, t)}{\partial t} \right]_{\vec{a}} &= \underbrace{\left( \frac{\partial F}{\partial t'} \right)_{\vec{r}'}}_{t=t'} \cdot \cancel{\left( \frac{\partial t'}{\partial t} \right)} \\
 &+ \underbrace{\left( \frac{\partial F}{\partial \vec{r}'} \right)_{t'}}_{\text{spatial gradient of } F} \cdot \underbrace{\cancel{\left( \frac{\partial \vec{r}'}{\partial \vec{r}} \right)}}_{\text{no frame rotation} \rightarrow I} \cdot \underbrace{\left( \frac{\partial \vec{r}}{\partial t} \right)_{\vec{a}}}_{=\vec{u}}
 \end{aligned}$$

- And, therefore:

$$\left( \frac{\partial F}{\partial t} \right)_a = \frac{\partial F}{\partial t'} + (\nabla' F) \cdot \vec{u} = \frac{DF}{Dt}$$

# ASIDE: Lagrangian Derivatives in an Eulerian Description (KC section 3.3)

- The result is the *Material Derivative*:

$$\left(\frac{\partial F}{\partial t}\right)_a = \frac{\partial F}{\partial t} + \vec{u} \cdot (\nabla F) = \frac{DF}{Dt}$$

- Which is a combination of:

- 1 The *local rate of change* of  $F$  w.r.t. time (ie. the change in  $F$  w.r.t. time as we sit at  $\vec{r}'$ ):

$$\frac{\partial F}{\partial t}$$

- 2 The *convective, or advective rate of change* of  $F$  w.r.t. time (ie. the change in  $F$  w.r.t. time due to the particle moving from one location to another, as measured in a field reference frame):

$$\vec{u} \cdot (\nabla F)$$

## ASIDE: Lagrangian Derivatives in an Eulerian Description(KC section 3.3)

- Consider a fluid with a known temperature gradient flowing in a tank.

## **ASIDE: Lagrangian Derivatives in an Eulerian Description(KC section 3.3)**

# Control Volume Integral Conservation Transport Equation

- Returning to our integral conservation law expression.
- Recall: We want to be able to calculate:

$$\frac{dB}{dt} = \iiint_{MV} \rho \frac{Db}{Dt} dV$$

using an *Eulerian description* of the fluid.

- Must invoke the *material derivative* for the acceleration term:

$$\frac{dB}{dt} = \iiint_{MV} \rho \left( \frac{\partial b}{\partial t} + (\vec{u} \cdot \nabla) b \right) dV$$

# Control Volume Integral Conservation Transport Equation

- Mathematical manipulation of the material derivative:

$$\begin{aligned}
 \rho \frac{Db}{Dt} &= \rho \frac{\partial b}{\partial t} + \rho (\vec{u} \cdot \nabla) b \\
 &= \rho \frac{\partial b}{\partial t} + \underbrace{\nabla \cdot (\rho \vec{u} b) - b \nabla \cdot (\rho \vec{u})}_{\nabla \cdot (a \vec{A}) = a(\nabla \cdot \vec{A}) + (\vec{A} \cdot \nabla) a : \text{Vector Ident.}} \\
 &= \rho \frac{\partial b}{\partial t} + \underbrace{\nabla \cdot (\rho \vec{u} b) - b \frac{\partial \rho}{\partial t}}_{\nabla \cdot (\rho \vec{u}) = -\frac{\partial \rho}{\partial t} : \text{Cons. of mass}} \\
 \rho \frac{Db}{Dt} &= \underbrace{\frac{\partial \rho b}{\partial t} + \nabla \cdot (\rho \vec{u} b)}_{\text{Eulerian Conservation Form (CFD)}}
 \end{aligned}$$



# ASIDE: Control Volume Integral Conservation Transport Equation

- This particular result is rather interesting:

$$\begin{aligned}\rho \frac{Db}{Dt} &= \rho \frac{\partial b}{\partial t} + \rho (\vec{u} \cdot \nabla) b \\ &= \frac{\partial \rho b}{\partial t} + \nabla \cdot (\rho \vec{u} b)\end{aligned}$$

- Why? The two equations represent the change of a property  $b$  with respect to time. (1) is the traditional material derivative statement, (2) indicates a conservative quality – that the change in the material property  $b$  has contributions due to changes of the property  $b$  in the differential volume and the divergence (or flux) of the property  $b$  through the CS (fascinating – we'll see this *template* equation later in the CFD lecture!)

# Control Volume Integral Conservation Transport Equation

- Back to the derivation:

$$\begin{aligned}\frac{dB}{dt} &= \iiint_{MV} \rho \left( \frac{\partial b}{\partial t} + (\vec{u} \cdot \nabla) b \right) dV \\ &\quad \text{From derivation on previous slide} \\ &= \overbrace{\iiint_{CV} \left( \frac{\partial(\rho b)}{\partial t} + \nabla \cdot (\rho \vec{u} b) \right) dV}^{\text{Gauss's Theorem, see aside}} \\ \frac{dB}{dt} &= \iiint_{CV} \frac{\partial(\rho b)}{\partial t} + \overbrace{\iint_{CS} \rho b (\vec{u} \cdot \hat{n}) dS}\end{aligned}$$

- This is a general statement of the time derivative of  $B$  with respect to  $t$  in a Material Volume (Lagrangian) in terms of an *Eulerian description*.

# Control Volume Integral Conservation Transport Equation

- Aside: Gauss's theorem:

# Control Volume Integral Conservation Transport Equation

- It is easier to deal with, however, if we pull the time rate of change out of the first integral (careful to apply Leibniz's Theorem correctly):

$$\frac{dB}{dt} = \frac{d}{dt} \iiint_{CV(t)} \rho b dV + \iint_{CS(t)} \rho b \left( (\vec{u} - \vec{V}_{CS}) \cdot \hat{n} \right) dA$$

- This is more general than the statement you had learned about in undergraduate texts which implies that the control volume to be fixed in space – which is a special case of this expression.

# Control Volume Integral Conservation Transport Equation

$$\underbrace{\frac{dB}{dt}}_{\text{Lagrangian}} = \underbrace{\frac{d}{dt} \iiint_{CV} \rho b dV}_{\text{Rate of change of } B \text{ inside } CV} + \underbrace{\iint_{CS} \rho b \left( (\vec{u} - \vec{V}_{CS}) \cdot \hat{n} \right) dA}_{\text{Flux of } B \text{ through } CS}$$

Property	$B$	$b$
Mass	$m_s$	1
Linear Momentum	$m_s \vec{u}$	$\vec{u}$
Angular Momentum	$\vec{r} \times m_s \vec{u}$	$\vec{r} \times \vec{u}$
Kinetic Energy	$\frac{1}{2} m_s \vec{u}^2$	$\frac{\vec{u}^2}{2}$

# Control Volume Integral Conservation Transport Equation

- Transport equation:

$$\frac{dB}{dt} = \frac{d}{dt} \iiint_{CV} \rho b dV + \iint_{CS} \rho b \left( (\vec{u} - \vec{V}_{CS}) \cdot \hat{n} \right) dA$$

- Conservation of Mass,  $b = 1$ :

$$\frac{dMass}{dt} \text{ Lagrangian} = 0 = \frac{d}{dt} \iiint_{CV} \rho dV + \iint_{CS} \rho \left( (\vec{u} - \vec{V}_{CS}) \cdot \hat{n} \right) dA$$

- The general transport theorem works for getting conservation of mass.

# Integral Conservation of Momentum for a Control Volume

- Transport equation:

$$\frac{dB}{dt} = \frac{\partial}{\partial t} \iiint_{CV} \rho b dV + \iint_{CS} \rho b \left( (\vec{u} - \vec{V}_{CS}) \cdot \hat{n} \right) dA$$

- Conservation of momentum,  $b = \vec{u}$  – momentum per unit mass:

$$\begin{aligned} \frac{d\vec{M}}{dt} &= \frac{\partial}{\partial t} \iiint_{CV} \rho \vec{u} dV + \iint_{CS} \rho \vec{u} \left( (\vec{u} - \vec{V}_{CS}) \cdot \hat{n} \right) dA \\ &= \iint_{CS} (-p\hat{n}) dS + \int \int_{CS} \tau dS + \iiint_{CV} \rho \vec{g} dV + \vec{F}_{applied} \end{aligned}$$

# Integral Conservation of Momentum for a Control Volume

Let's pay some attention to the RHS of the equation and what it says:

$$= \iint_{CS} (-p\hat{n})dS + \iint_{CS} \tau dS + \iiint_{CV} \rho\vec{g}dV + \vec{F}_{applied}$$

The conservation of linear momentum expression says that the **rate of change of momentum** is equal to the **sum of forces applied to the control volume**. Some tips:

- Whenever a solid surface is cut by the control surface, an external force should be applied to the control volume at that point.
- If in doubt, *always* draw and account for the pressure (normal stress) and shear stress on all sides of the control surface.
- The mass and any body forces should be accounted for in the problem.



# Integral Conservation of Momentum for a Control Volume

- Be careful to see what the momentum equation means.
  - 1 It is a vector equation – one equation for each *momentum/force* direction
  - 2 Tricky: The normal velocity out of the control surface may not be the same direction as the momentum flux out of that control volume – see example.
  - 3 The forces causing the change in momentum can be volume forces, surface forces, or point loads. The total effect of all of these forces on the CV/CS must equate to the change in momentum.

$$\frac{\partial}{\partial t} \iiint_{CV} \rho u_x dV + \iint_{CS} \rho u_x \left( (\vec{u} - \vec{V}_{CS}) \cdot \hat{n} \right) dA = \sum F_{Tx} \rightarrow x_{direction}$$

$$\frac{\partial}{\partial t} \iiint_{CV} \rho u_y dV + \iint_{CS} \rho u_y \left( (\vec{u} - \vec{V}_{CS}) \cdot \hat{n} \right) dA = \sum F_{Ty} \rightarrow y_{direction}$$

$$\frac{\partial}{\partial t} \iiint_{CV} \rho u_z dV + \iint_{CS} \rho u_z \left( (\vec{u} - \vec{V}_{CS}) \cdot \hat{n} \right) dA = \sum F_{Tz} \rightarrow z_{direction}$$

# Integral Conservation of Momentum for a Control Volume: Examples

- Conservation of Mass,  $b = 1$ :

$$\frac{dMass}{dt} \text{ Lagrangian} = 0 = \frac{d}{dt} \iiint_{CV} \rho dV + \iint_{CS} \rho \left( (\vec{u} - \vec{V}_{CS}) \cdot \hat{n} \right) dA$$

- Conservation of momentum,  $b = \vec{u}$  – momentum per unit mass:

$$\begin{aligned} \frac{d\vec{M}}{dt} &= \frac{\partial}{\partial t} \iiint_{CV} \rho \vec{u} dV + \iint_{CS} \rho \vec{u} \left( (\vec{u} - \vec{V}_{CS}) \cdot \hat{n} \right) dA \\ &= \iint_{CS} (-p\hat{n}) dS + \int \int_{CS} \tau dS + \iiint_{CV} \rho \vec{g} dV + \vec{F}_{ap} \\ &= \sum \vec{F}_T \end{aligned}$$